

# Last Lecture:

LAST = Previous

E1

INTRODUCED PLANE KINETICS OF RIGID BODIES

$$\underline{\underline{\sum \vec{F} = m \vec{a}_G}}$$

$$\underline{\underline{\sum \vec{M}_G = \dot{H}_G = \alpha I_G}}$$

(FOR 2-D CASE)

⊗  $I_G = \int r^2 dm$  moment of inertia  
 $= \rho \int r^2 dV$  if  $\rho$  is constant throughout volume

Parallel axis theorem:  $I = I_G + md^2$

Radius of Gyration:  $k = \sqrt{\frac{I}{m}} \Leftrightarrow I = k^2 m$

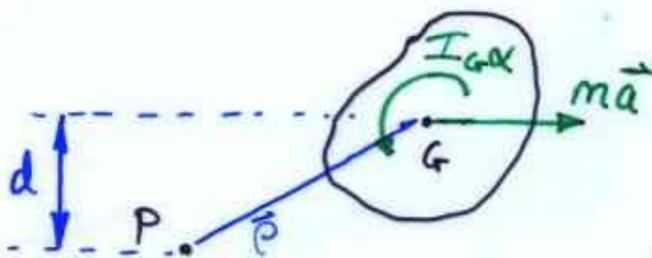
Consider alternative forms of moment eqn..

$$\sum \vec{M}_P = \dot{H}_G + \vec{r} \times m \vec{a}_G \quad \vec{r} \text{ is vector } \vec{PG}$$

or 2D:

$$\sum M_P = I_G \alpha \pm m a_G d$$

$d = \perp$  dist between  
P & vector of accel  
sign depends on  
right hand rule.



Here  $\sum M_P = +I_G \alpha - m a d$  NOTE sign

P is a nonaccelerating point

Another form is:

E(2)

$$\sum \vec{M}_P = I_P \alpha + \vec{r} \times m \vec{a}_P$$

Here  $P$  is a point fixed to the body

$\vec{a}_P$  is accel of  $P$

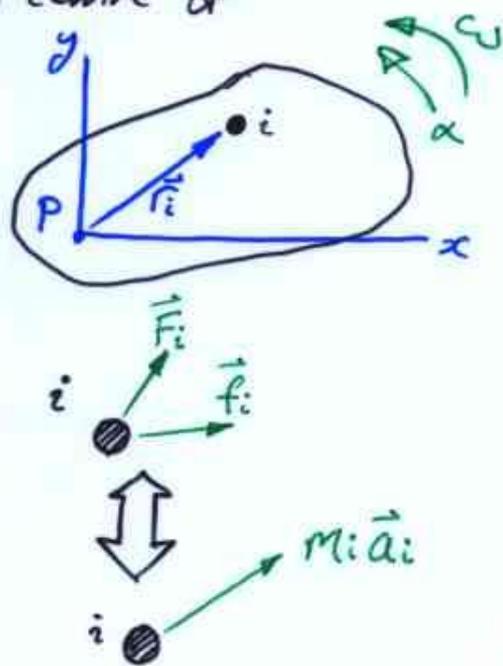
$\vec{r}$  is vector from  $P$  to mass centre  $G$

Consider small particle  $i$

$\vec{F}_i$  = Resultant External force acting on particle

$\vec{f}_i$  = resultant internal force due to interactions with adjacent particles

mass is  $m_i$



Sum moments about P:

$$\vec{r}_i \times \vec{F}_i + \vec{r}_i \times \vec{f}_i = \vec{r}_i \times m_i \vec{a}_i$$

i.e.  $\vec{M}_{P_i} = \vec{r}_i \times m_i \vec{a}_i$

in general, Point P can have an acceleration

$$\Rightarrow \vec{a}_i = \vec{a}_P + \vec{\alpha} \times \vec{r}_i - \omega^2 \vec{r}_i$$

so  $\vec{M}_{P_i} = m_i \vec{r}_i \times (\vec{a}_P + \vec{\alpha} \times \vec{r}_i - \omega^2 \vec{r}_i)$

$$= m_i [\vec{r}_i \times \vec{a}_P + \vec{r}_i \times (\vec{\alpha} \times \vec{r}_i) - 0]$$

$$\boxed{\vec{r}_i \times \vec{r}_i = 0}$$



2D problem ... So:

E (3)

$$\vec{r}_i = x_i \hat{i} + y_i \hat{j} \quad \vec{a}_P = a_{Px} \hat{i} + a_{Py} \hat{j}$$

$$\vec{\alpha} = \alpha \hat{k} \quad \text{and} \quad \vec{M}_{Pi} = M_{Pi} \hat{k}$$

$$\text{So } M_{Pi} \hat{k} = m_i \left\{ (x_i \hat{i} + y_i \hat{j}) \times [a_{Px} \hat{i} + a_{Py} \hat{j}] + (x_i \hat{i} + y_i \hat{j}) \times [\alpha \hat{k} \times (x_i \hat{i} + y_i \hat{j})] \right\}$$

$$M_{Pi} \hat{k} = m_i \left\{ -y_i a_{Px} + x_i a_{Py} + \alpha (x_i^2 + y_i^2) \right\} \hat{k}$$

note  $x^2 + y^2 = r^2$

if we let  $m_i \rightarrow dm$  & integrate over mass  $m$  we get ...

$$\sum M_P = - \left( \int_m y dm \right) a_{Px} + \left( \int_m x dm \right) a_{Py} + \left( \int_m r^2 dm \right) \alpha$$

<p style="text-align: center;">↓</p> <p>moment of external forces about Point P (internal moments cancel out)</p>	<p style="text-align: center;">↓</p> <p><math>m y_G</math> i.e. the integrals locate centre of mass</p>	<p style="text-align: center;">↓</p> <p><math>m x_G</math></p>	<p style="text-align: center;">↓</p> <p>M.O. Inertia about P <math>I_P</math></p>
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$$\sum M_P = m \vec{p} \times \vec{a}_P + I_P \alpha$$

$$\vec{p} = x_G \hat{i} + y_G \hat{j}$$

Say  $P = G$  (center of mass)

$$\Rightarrow \underline{\underline{\sum M_G = m \vec{0} \times \vec{a}_G + I_G \alpha = I_G \alpha}}$$

OR Say P has no accel

$$\Rightarrow \sum M_P = m \vec{p} \times \vec{0} + I_P \alpha$$

$$\underline{\underline{\sum M_P = I_P \alpha}}$$

# EQUATIONS OF MOTION

TRANSLATION: i.e. all parts of a body

undergoing TRANSLATION have the same acceleration.

⇒ no rotation about c.o. gravity

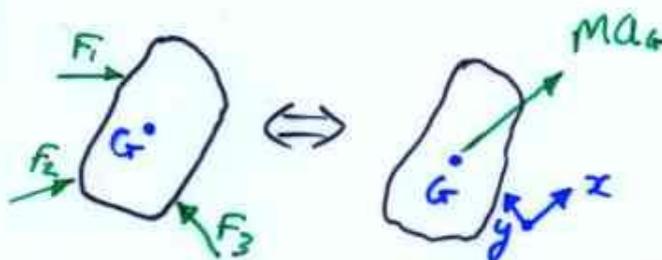
⇒  $\sum M_G = 0$  and of course:  $\sum \vec{F} = M\vec{a}_G$

TRANSLATION may be RECTILINEAR or CURVILINEAR

## RECTILINEAR Translation:

all points move in straight lines

CONVENIENT TO PUT x-axis parallel to  $\vec{a}$

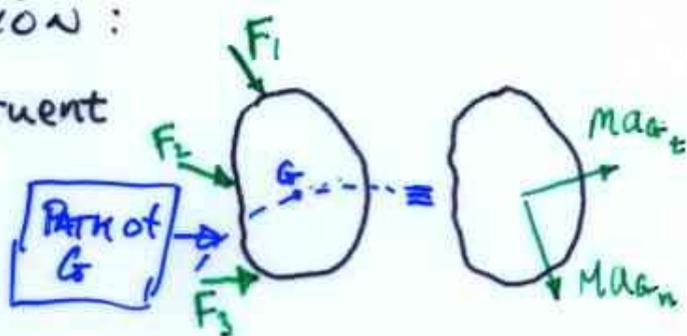


## CURVILINEAR Translation:

all points move on congruent curved paths

convenient to use components normal & tangential to path of G

& tangential to path of G



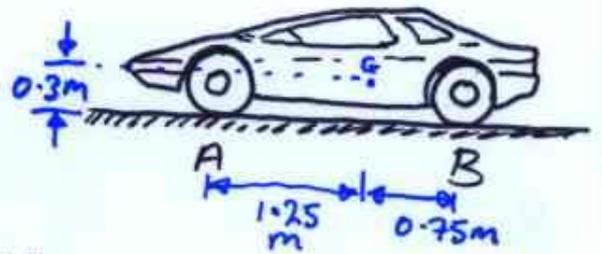
$$\sum M_G = 0, \quad \sum F_n = M a_{Gn}, \quad \sum F_t = M a_{Gt}$$

Note that other points can be used as moment centres?

# Example

E(5)

car has mass 2000 kg  
REAR WHEEL DRIVE  
CALCULATE accln if rear WHEELS  
ARE SLIPPING & FRONT ROTATE FREELY

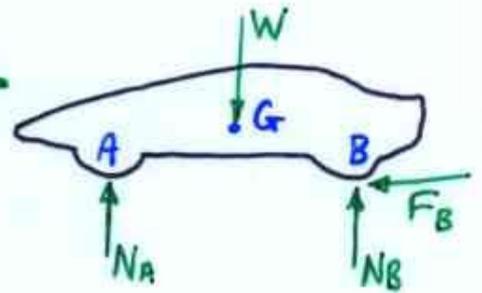


coeff of kinetic friction  $\mu_k = 0.25$

DRAW FREE BODY DIAGRAM

$$W = mg = (2000)(9.81) = 19.62 \text{ kN}$$

$$F_B = \mu_k N_B \quad \sum F_y = ma_y = m(0)$$



$$\uparrow \sum F_y = 0 \Rightarrow N_A + N_B - W = 0 \quad (1)$$

$$\leftarrow \sum F_x = ma_{Gx} \Rightarrow \mu_k N_B = 2000 a_{Gx} \quad (2)$$

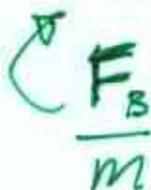
$$\oplus \sum M_G = 0 \Rightarrow -1.25 N_A + 0.75 N_B - (0.3) \mu_k N_B = 0 \quad (3)$$

Substitute (1) into (3) for ~~N\_B~~  $N_A$

$$\Rightarrow -1.25 [W - N_B] + 0.75 N_B - 0.3 \mu_k N_B = 0$$

$$\Rightarrow N_B = \frac{1.25 W}{1.25 + 0.75 - 0.3 \mu_k} = \frac{(1.25)(2000)(9.81)}{1.25 + 0.75 - (0.3)(0.25)} = 12740 \text{ N}$$

$$a_{Gx} = \frac{\mu_k N_B}{m} = \frac{(0.25)(12740)}{(2000)} = \underline{1.59 \text{ m/s}^2}$$

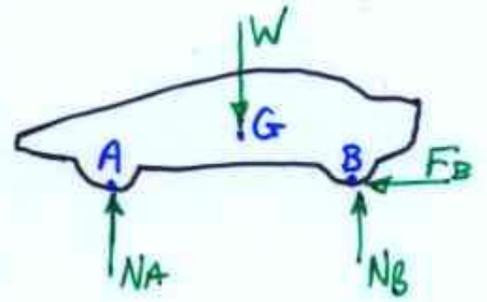


There was an <sup>SLIGHTLY</sup> easier way to solve problem E(6)

TAKE MOMENTS ABOUT POINT A  
then  $\vec{N}_A$  &  $\vec{F}_B$  HAVE ZERO MOMENT

$$\oplus \sum M_A = ma_{Gx}d$$

$$-W(1.25) + N_B(2.0) = +ma_{Gx}(0.3)$$



using ② from before i.e.  $N_B = \frac{2000 a_{Gx}}{\mu_k}$

$$\text{gives } -W(1.25) + \left(\frac{2000 a_{Gx}}{\mu_k}\right)(2.0) = +(ma_{Gx})(0.3)$$

$$\Rightarrow a_{Gx} = \frac{W(1.25)}{\left[\frac{(2000)(2.0)}{\mu_k}\right] - (2000)(0.3)} = \frac{(2000)(9.81)(1.25)}{\frac{(2000)(2.0)}{\mu_k} - (2000)(0.3)}$$

$$\underline{a_{Gx} = 1.5925 \text{ m s}^{-2}}$$

Note HOW  $a_{Gx}$  is  
INDEPENDENT of  $m$   
(i.e. we have "2000" above & below)

$a_{Gx}$  is a fn of "g" & geometry &  $\mu_k$ .

6/33 mass of crate = 900 kg

Rope at C is cut. FIND tension in cable at A in instant immediately after. Solve using 3 EQNS, and using just 1.

$\theta = 60^\circ$



JUST AFTER ROPE CUT, CRATE IS STILL AT REST.

$\Rightarrow \omega = 0$  so  $r\omega^2$  component

of acceleration is zero

$\Rightarrow a = r\ddot{\theta}$  and is tangential to path of travel. (as shown in green)

Use components n, t (normal, tangential)

$\sum F_n = ma_n = 0 = r\omega^2$

$\Rightarrow T_A + T_B - W_n = 0$

OR  $T_A + T_B = mg \cos(30^\circ)$  \*

$\sum F_t = ma_t$

$\Rightarrow W_t = ma_t \Rightarrow a = \frac{mg \sin(30^\circ)}{m} = \frac{g}{2} \text{ ms}^{-2}$

$\sum M_G = 0$

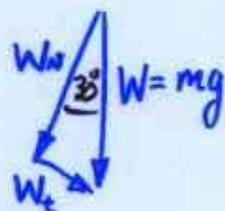
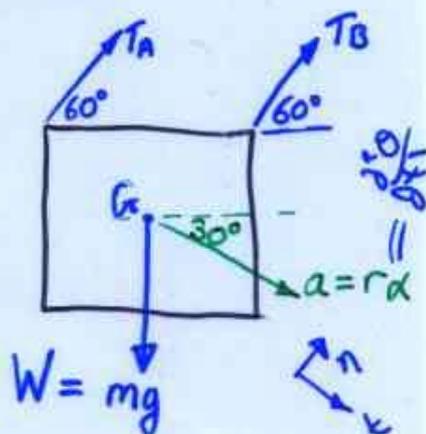
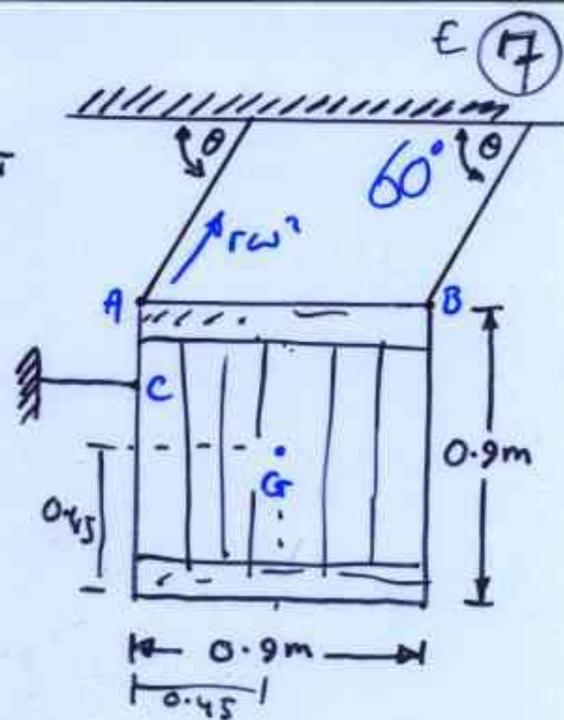
$\Rightarrow T_B \left[ \frac{0.9}{\sqrt{2}} \sin(15^\circ) \right] - \left[ T_A \left( \frac{0.9}{\sqrt{2}} \right) \cos(15^\circ) \right] = 0$

$\Rightarrow T_B = 3.7273 T_A$

Substitute into 1st Expression \*

$T_A + 3.7273 T_A = (900)(9.81)(\cos 30^\circ)$

$\Rightarrow T_A = 1617 \text{ N}$



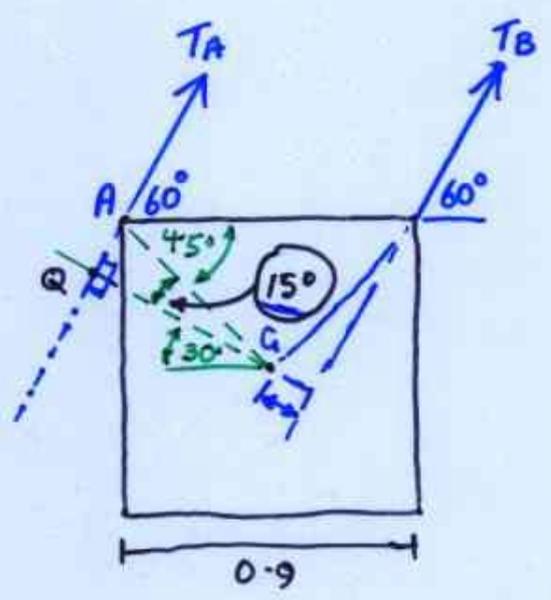
$a_n = 0$   
 $\Rightarrow a = a_t$

$\vec{W}$  has no moment about G

Note  $|AG| = (0.9) \cos 45^\circ$   
 $= \frac{0.9}{\sqrt{2}}$

$\Rightarrow |GQ| = \left(\frac{0.9}{\sqrt{2}}\right) \cos(15^\circ)$

MOMENT ARM  
of  $\vec{T}_A$



MORE DIRECT APPROACH

take moment about P  
i.e. intersection of  $\vec{T}_B$  &  $\vec{a}$

$\Sigma M_P = I_G \alpha + m a_G d$

$I_G \alpha = 0$  because TRANSLATION  
 $\Rightarrow \alpha = 0$

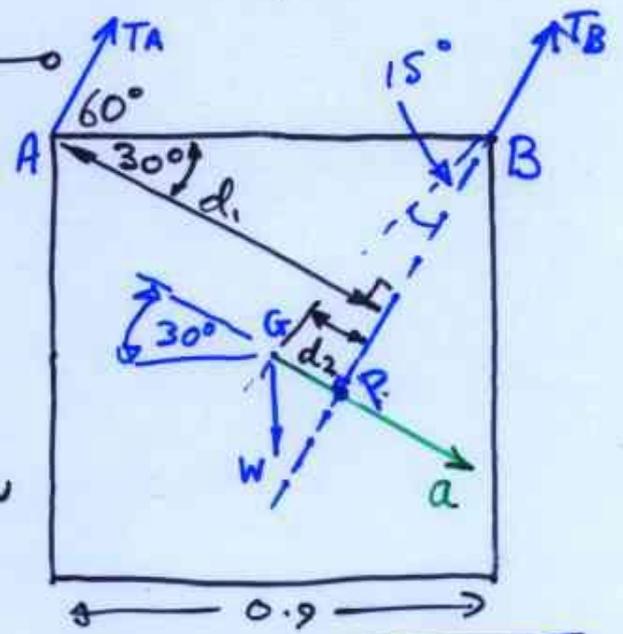
$m a_G d = 0$  because  $d = 0$   
 due to location of P

$\therefore \Sigma M_P = 0$

$\Rightarrow -T_A d_1 + W d_2 = 0$

$\Leftrightarrow T_A = \frac{W d_2}{d_1}$   
 $T_A = \frac{(m \times g) \cos(30^\circ) \left(\frac{0.9}{\sqrt{2}}\right) \sin(15^\circ)}{(0.9) \cos(30^\circ)}$

$T_A = 1616 \text{ N}$



$d_1 = (0.9) (\cos 30^\circ)$

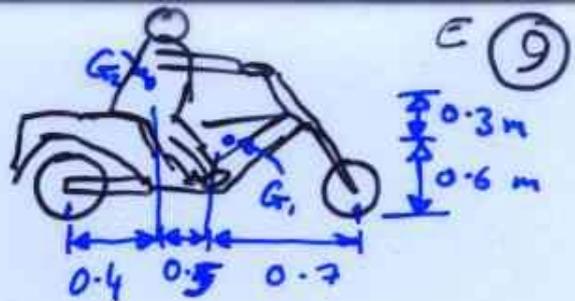
$d_2 = |PG|$   
 $|BG| = \frac{0.9}{\sqrt{2}}$   
 $\Rightarrow |PG| = \frac{0.9}{\sqrt{2}} \sin(15^\circ)$

Motorcycle

MASS = 125 kg C.O.G =  $G_1$

RIDER

MASS = 75 kg C.O.G =  $G_2$



Coeff of friction wheel  $\rightarrow$  ROAD =  $0.8 = \mu_s$

CALCULATE if a wheelie is possible  
& if so @ WHAT ACCLN it will occur

Assume mass of wheels small & front wheel free to roll.

First, FIND centre of mass

$$x\text{-coord} = \frac{(125)(0.9) + (75)(0.4)}{200} = 0.7125 \text{ m}$$

$$y\text{-coord} = \frac{(125)(0.6) + (75)(0.9)}{200} = 0.7125 \text{ m}$$

Both relative to B

$$\sum F_x = m a_{Gx} \Rightarrow F_B = m a_G \quad (1)$$

$$\sum F_y = m a_{Gy} = 0 \Rightarrow N_B + N_A - W = 0 \quad (2)$$

$$\sum M_B = m a_{Gy} \Rightarrow (N_A)(1.6) - (W)(0.7125) = -m a_G (0.7125) \quad (3)$$

IF A wheelie is about to occur  $\Rightarrow N_A = 0$

$$(2) \Rightarrow N_B = W = (200)(9.81)$$

$$(3) \Rightarrow (0.7125)(200)(9.81) = +(200)(a_G)(0.7125)$$
$$\Rightarrow a_G = 9.81 \text{ m s}^{-2}$$

$$(1) \Rightarrow F_B = m a_G = (200)(9.81)$$

So we have  $F_B = (200)(9.81)$  &  $N_B = (200)(9.81)$

BUT,  $\mu_s = 0.8 \Rightarrow$  MAX value of  $F_B$  is  $(0.8)(N_B)$

$\therefore$  wheelie CANNOT occur.

i.e. WHEEL WILL SKID BEFORE A WHEELIE WILL OCCUR.

E (10)

So... WHAT IS MAX ACCLN POSSIBLE?

Just before wheel spin

$$F_B = \mu_s N_B$$

$$\mu_s = 0.8$$

from ①  $N_A = W - N_B = mg - N_B$

from ②  $ma_G = F_B = \mu_s N_B$

then ③  $(N_A)(1.6) - W(0.7125) = -ma_G(0.7125)$

Becomes

$$(mg - N_B)(1.6) - mg(0.7125) = -\mu_s N_B (0.7125)$$

$$\Rightarrow N_B = mg \left[ \frac{0.7125 - 1.6}{-1.6 + (0.8)(0.7125)} \right] = 1690 \text{ N}$$

$$\therefore F_B = \mu_s N_B = (0.8)(1690 \text{ N}) = 1352 \text{ N}$$

$$\therefore a_G = \frac{F_B}{m} = \frac{1352}{200} = \underline{\underline{6.76 \text{ m/s}^2}}$$

NOTE, AT THIS POINT

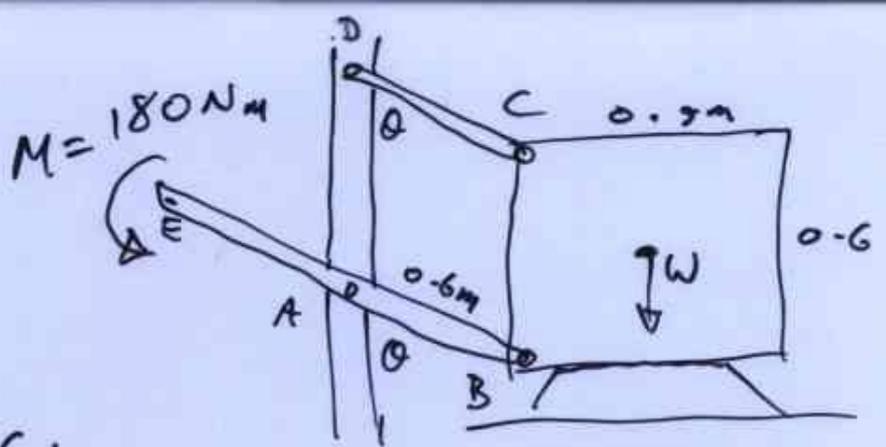
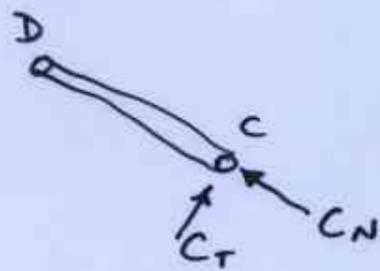
$$N_A = mg - N_B$$

$$= (200)(9.81) - 1690 = 272 \text{ N}$$

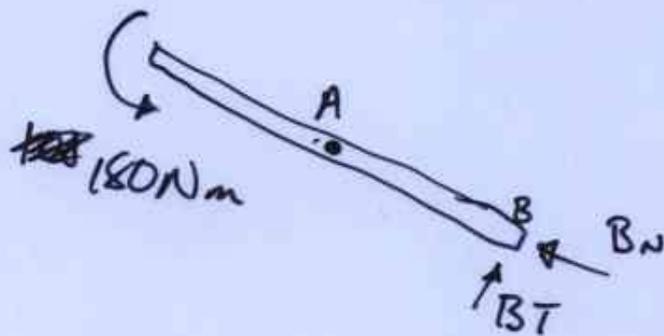
Max accln &  $N_A$  still  $> 0$

$\Rightarrow$  wheelie impossible

$$\theta = 60^\circ$$



Light Linkage  $\Rightarrow C_T = 0$



$$- |AB| B_T = +180 \text{ Nm}$$

$$\sum M = 0$$

Because LINK LIGHT

$$B_T = \frac{180}{|AB|} = \frac{180}{0.6} = 300 \text{ N}$$